



**JYOTHISHMATHI INSTITUTE OF TECHNOLOGY & SCIENCE**

**Nustulapur, Karimnagar - 505481**

**(Approved by AICTE, New Delhi & Affiliated to JNTUH)**

## **MECHANICS OF SOLIDS**

### **STRESS AND STRAIN IN BEAMS**



**BY**

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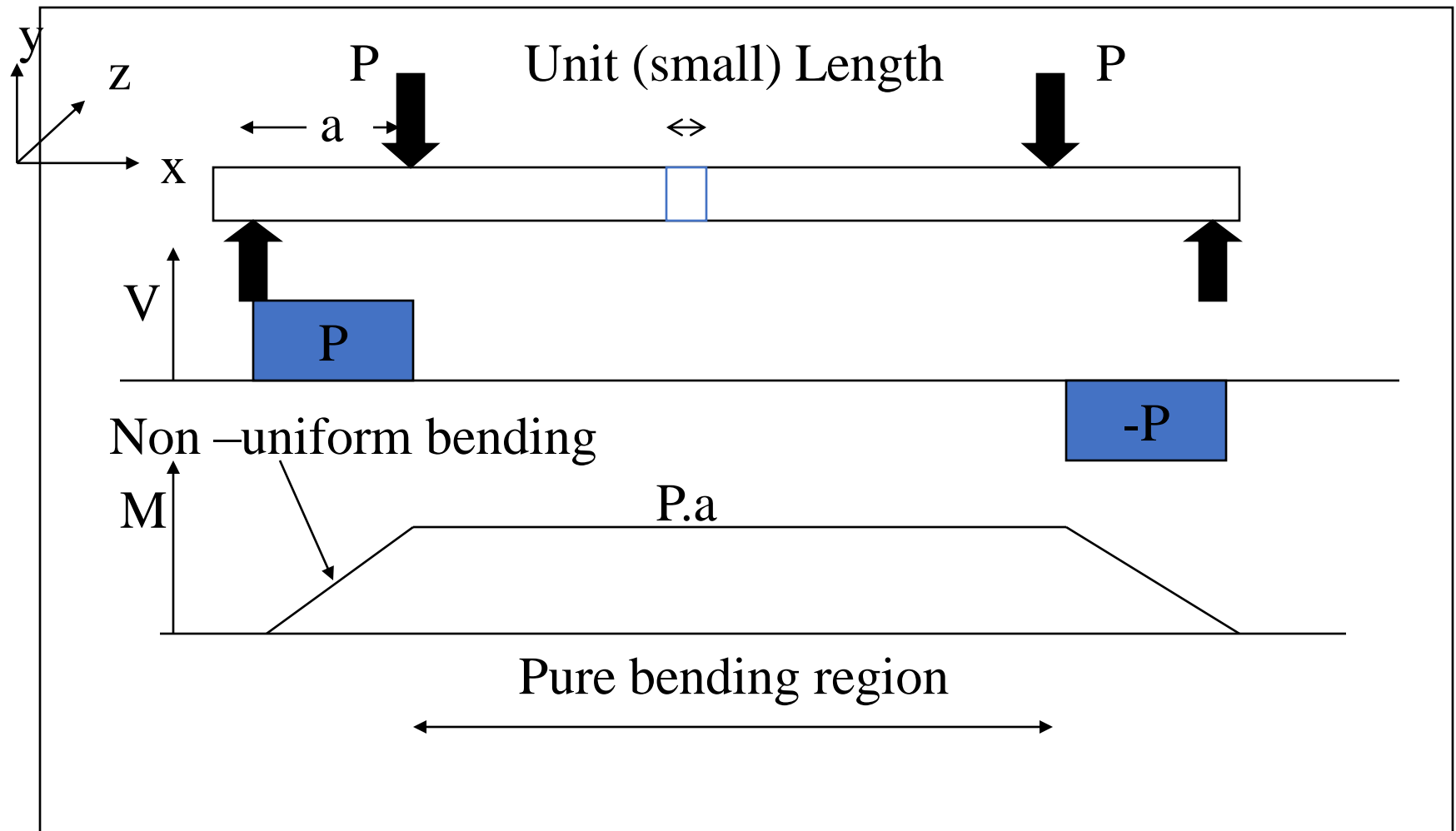
# BENDING BEAMS

- LOADS ON BEAM PRODUCE STRESS RESULTANTS,  $V$  &  $M$
- $V$  &  $M$  PRODUCE NORMAL STRESSES AND STRAINS IN PURE BENDING
- $V$  &  $M$  PRODUCE ADDITIONAL SHEAR STRESSES IN NON-UNIFORM BENDING

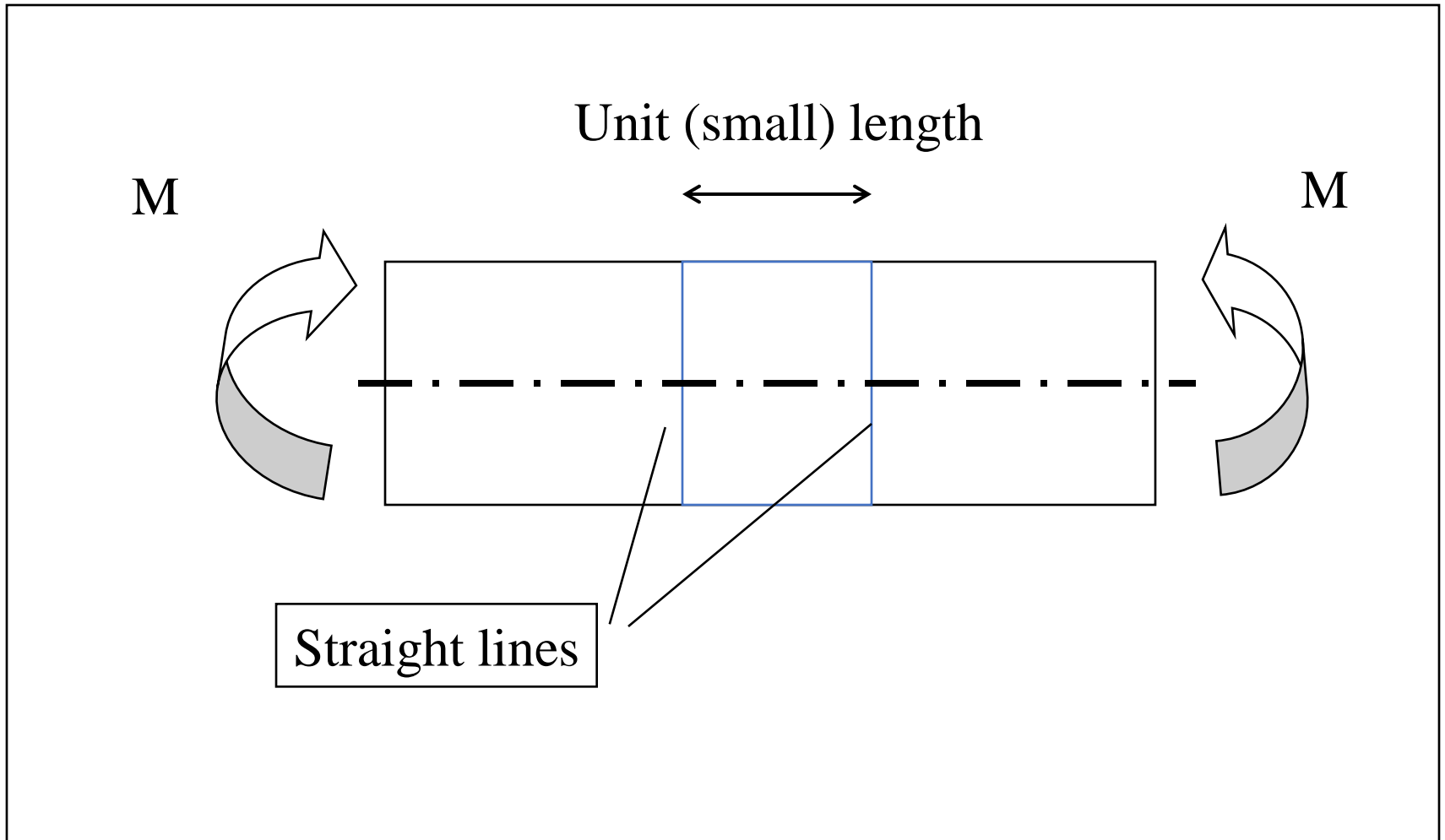
# TYPES OF BENDING

- PURE BENDING – FLEXURE UNDER CONSTANT M. i.e.  $V = 0 = dM/dx$
- NON-UNIFORM BENDING – FLEXURE WHEN V NON-ZERO

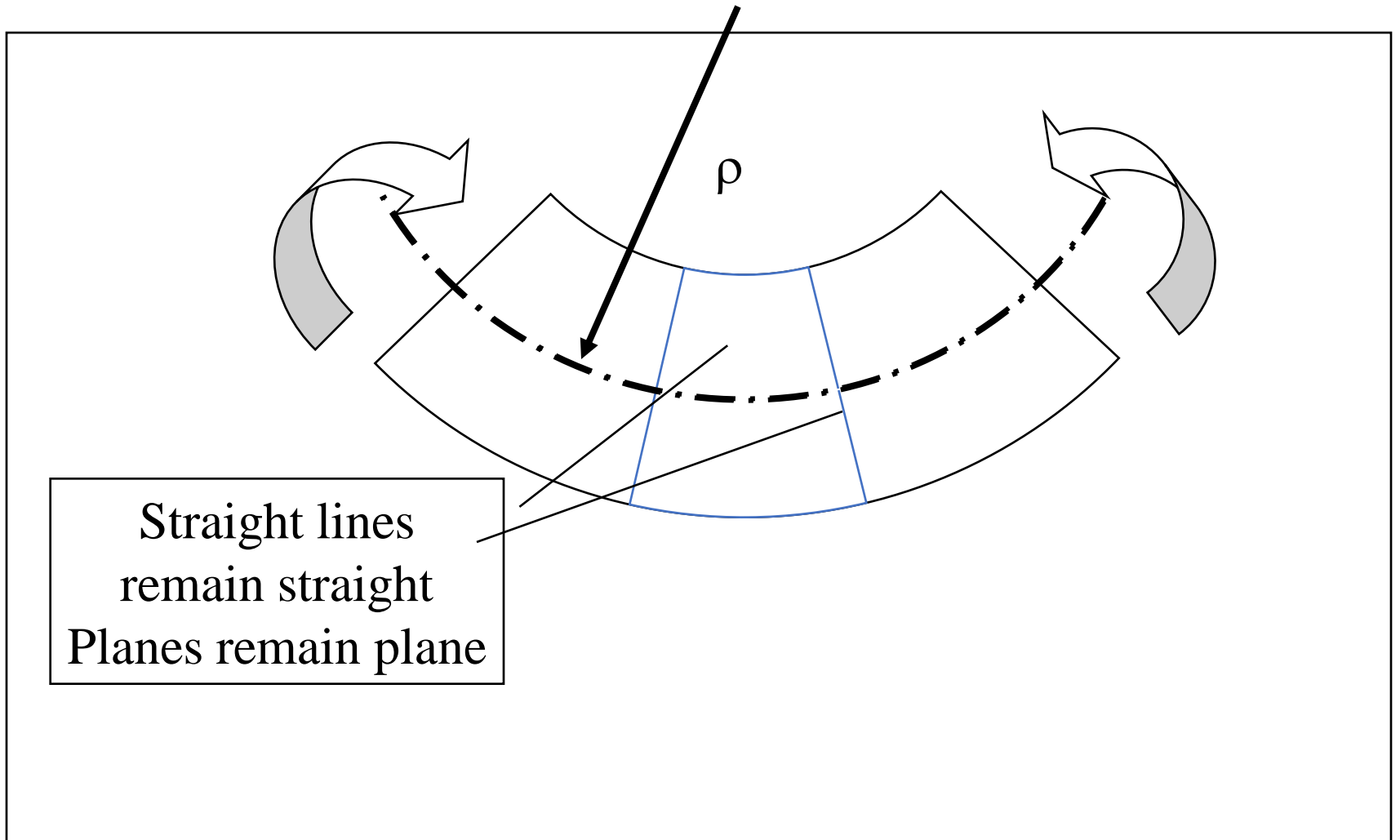
# PURE BENDING



# PURE BENDING

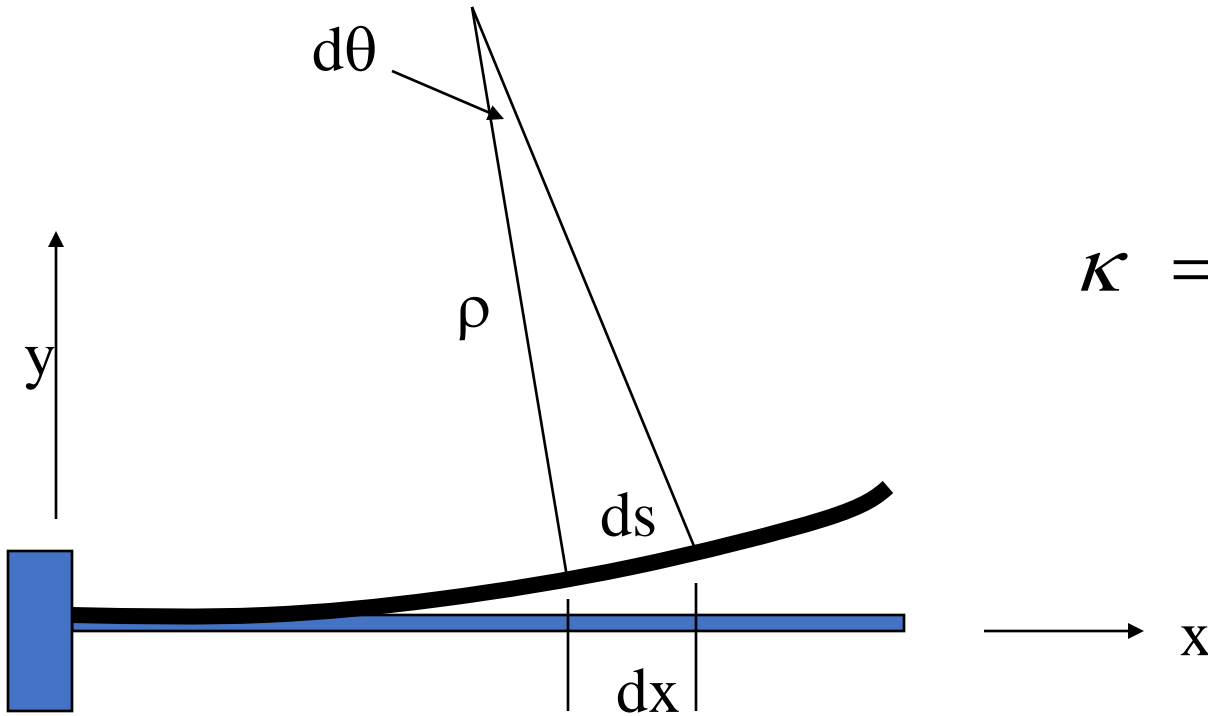


# RADIUS OF CURVATURE $\rho$



CURVATURE  $\kappa = 1/\rho$

$$\rho \cdot d\theta = ds = dx$$



$$\kappa = \frac{1}{\rho} = \frac{d\theta}{dx}$$

$$\rho d\theta = dx$$

$$L_1 = (\rho - y)d\theta$$

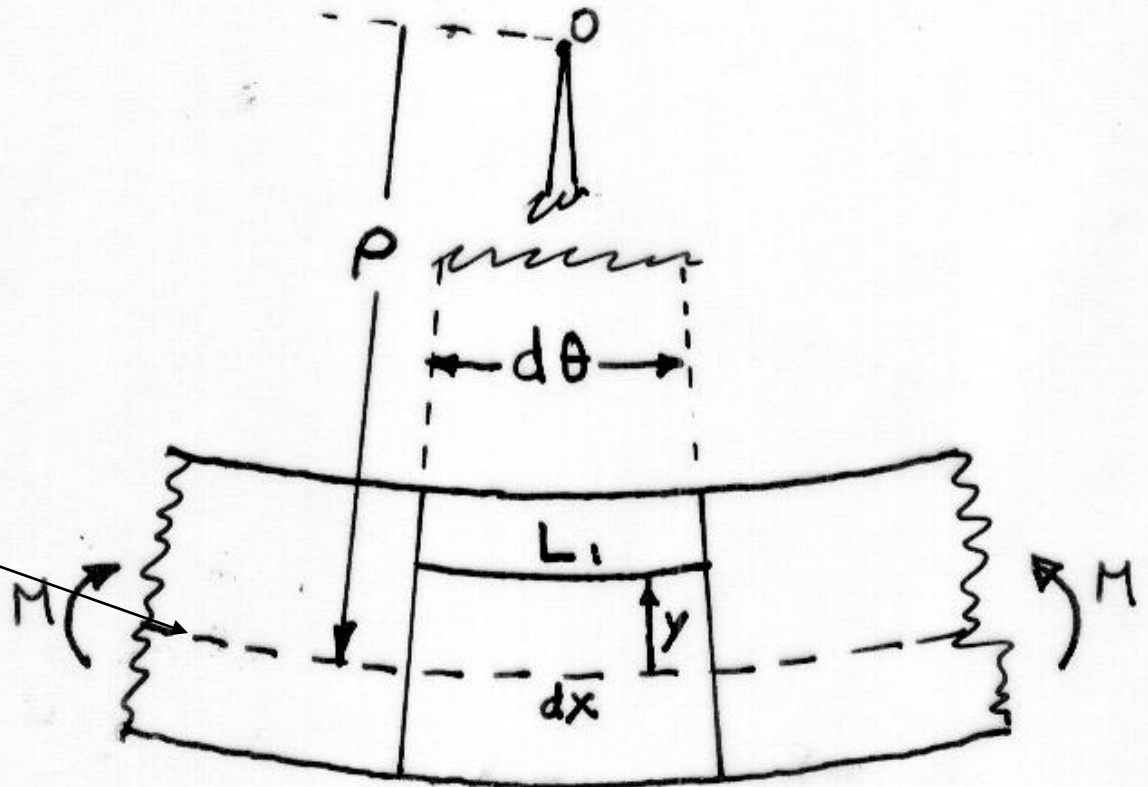
$$L_1 = dx - (y/\rho)dx$$

$$L_1 - dx = -(y/\rho)dx$$

NORMAL STRAIN

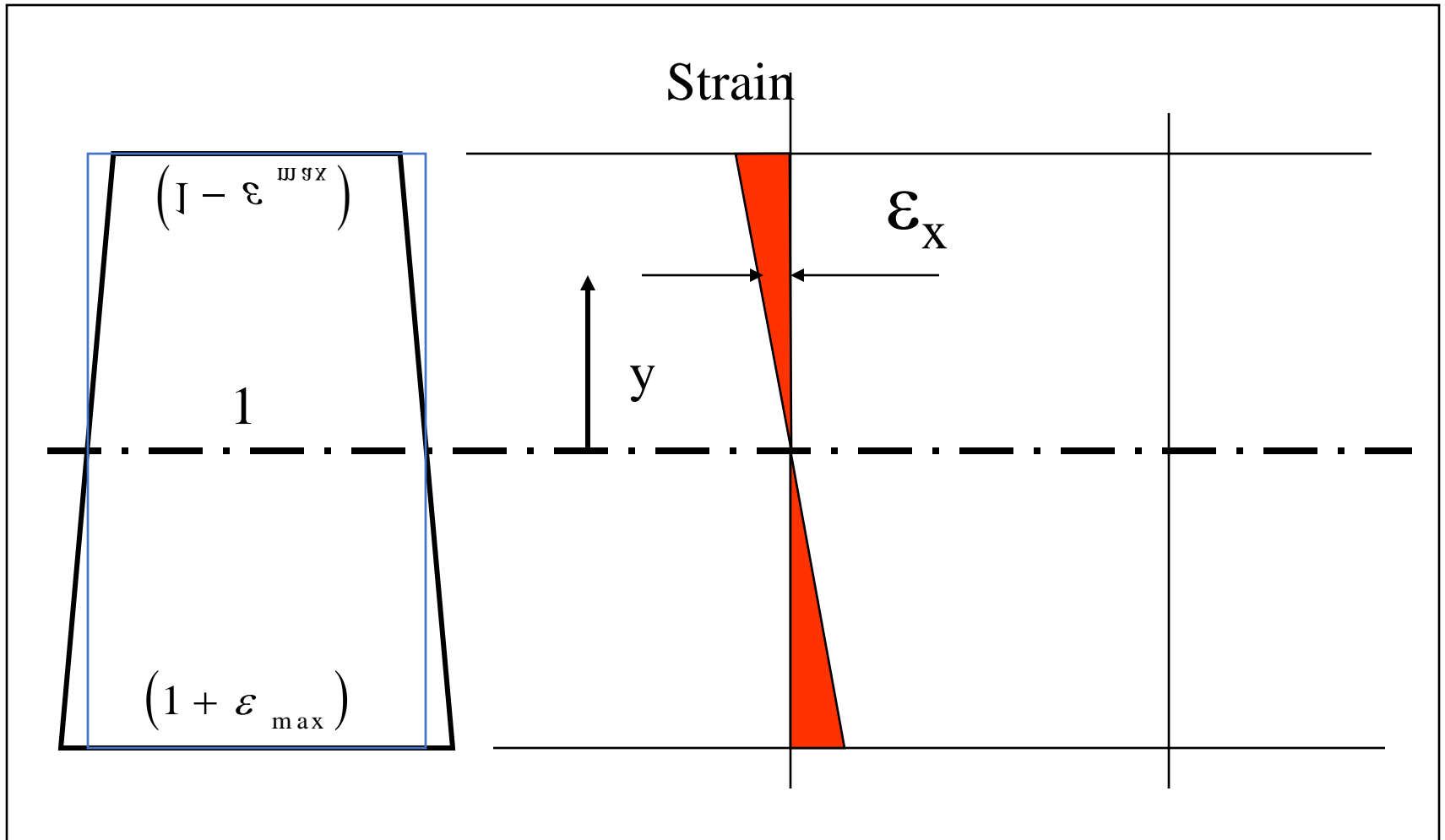
$$\varepsilon_x = -(y/\rho)dx/dx = -\kappa \cdot y$$

NEUTRAL AXIS

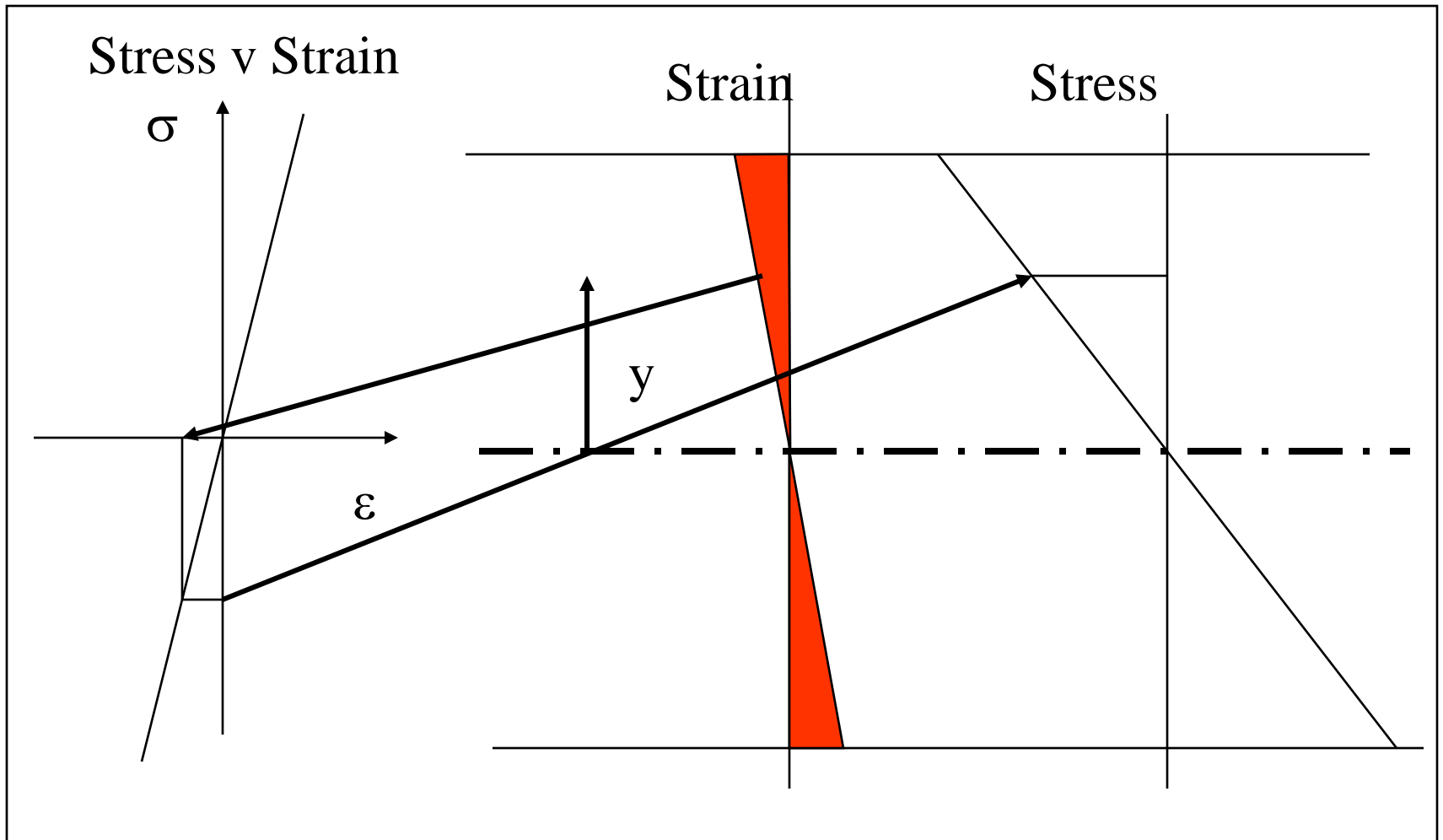




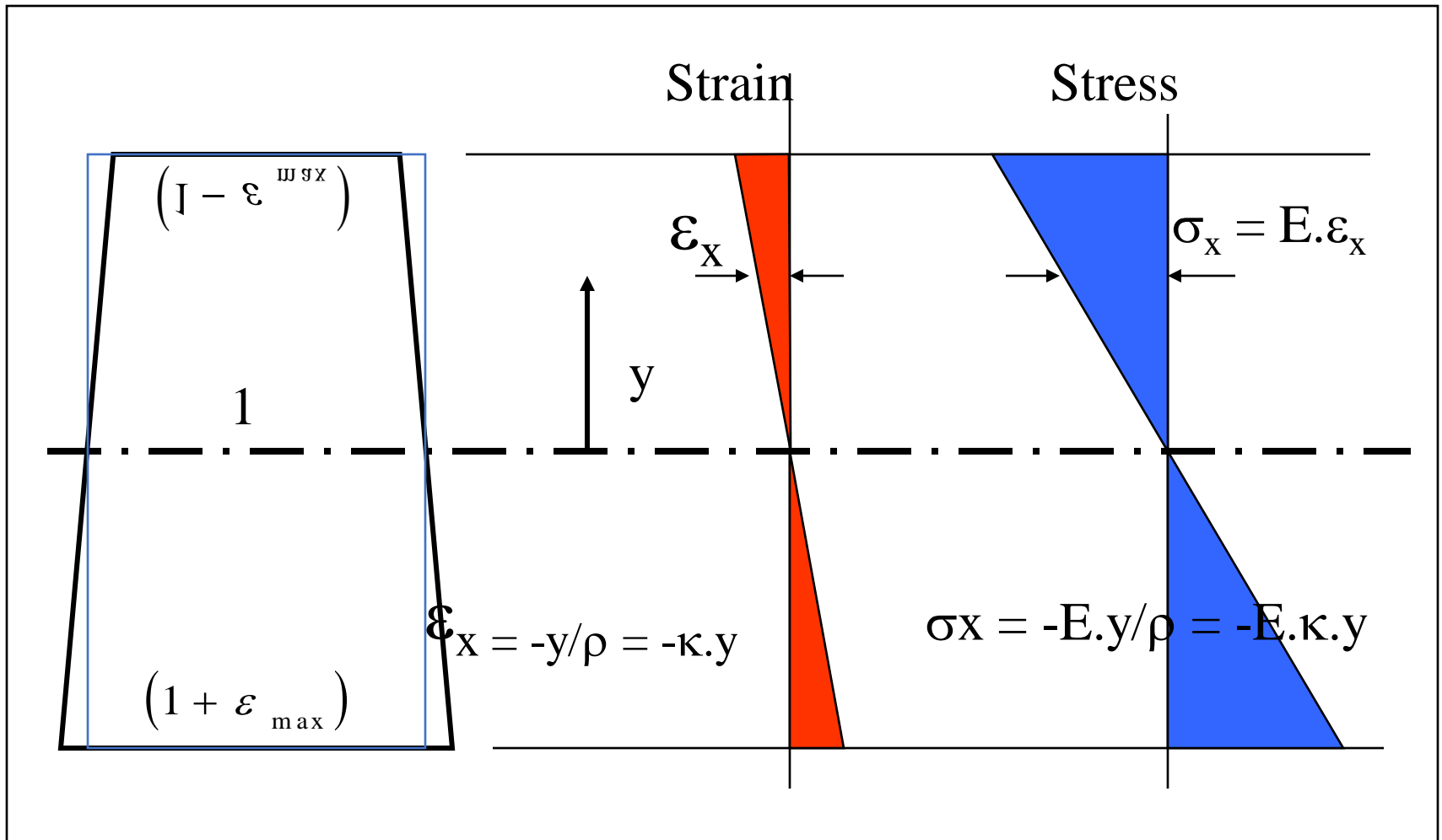
# LONGITUDINAL STRAIN



# LONGITUDINAL STRESS



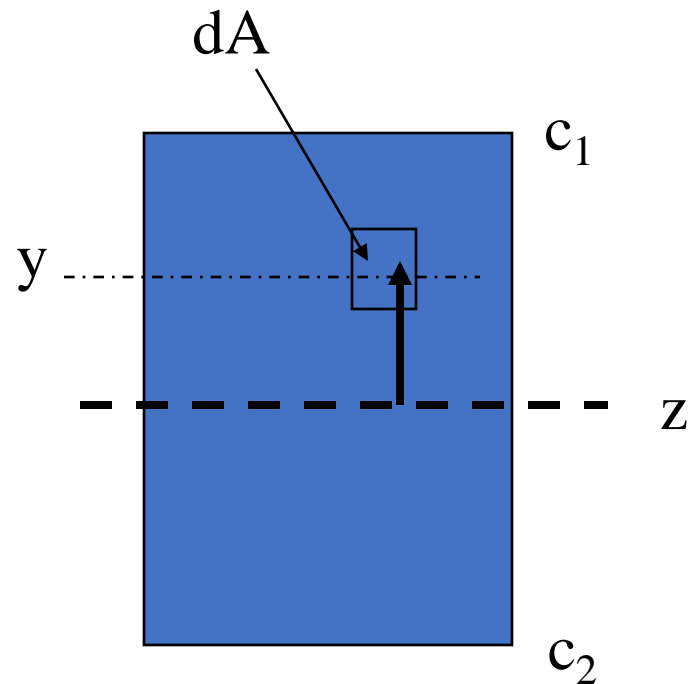
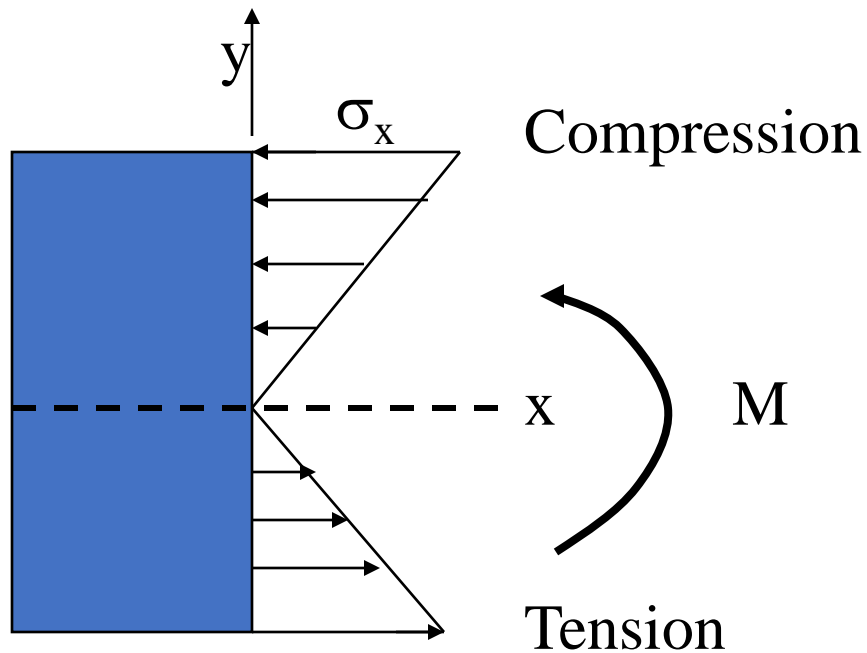
# NORMAL STRESS AND STRAIN



# RELATION BETWEEN $\sigma$ & $M$

- NEED TO KNOW WHERE NEUTRAL AXIS IS. FIND USING HORIZONTAL FORCE BALANCE
- NEED A RELATION BETWEEN  $\kappa$  &  $M$ . FIND USING MOMENT BALANCE  
–gives THE MOMENT CURVATURE EQUATION & THE FLEXURE FORMULA

# NEUTRAL AXIS PASSES THROUGH CENTROID



$$\int_A \sigma_x dA = - \int_A E \cdot \kappa \cdot y \cdot dA = 0$$

$$\int_A y \cdot dA = 0$$

First moment of area

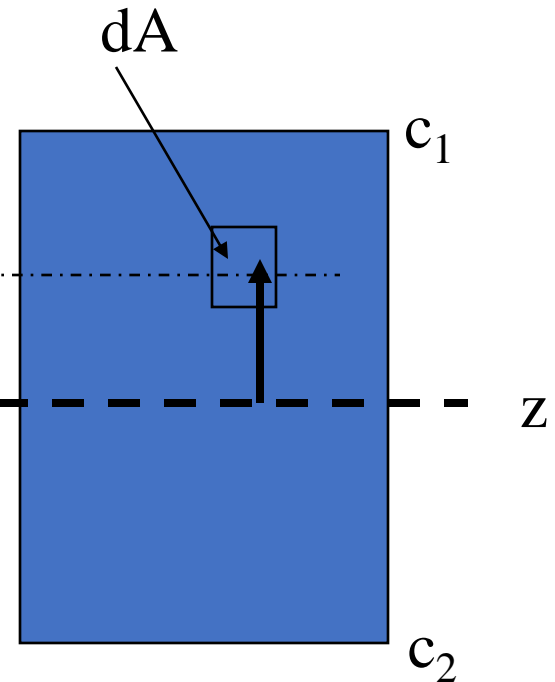
# MOMENT-CURVATURE EQN

$$dM = -\sigma_x y dA$$

$$M = -\int_A \sigma_x y dA = \int_A \kappa \cdot E \cdot y^2 dA$$

$$I = \int_A y^2 dA$$

= moment of inertia  
wrt neutral axis



$$M = \kappa \cdot E \cdot I$$

# FLEXURE FORMULA FOR BENDING STRESSES

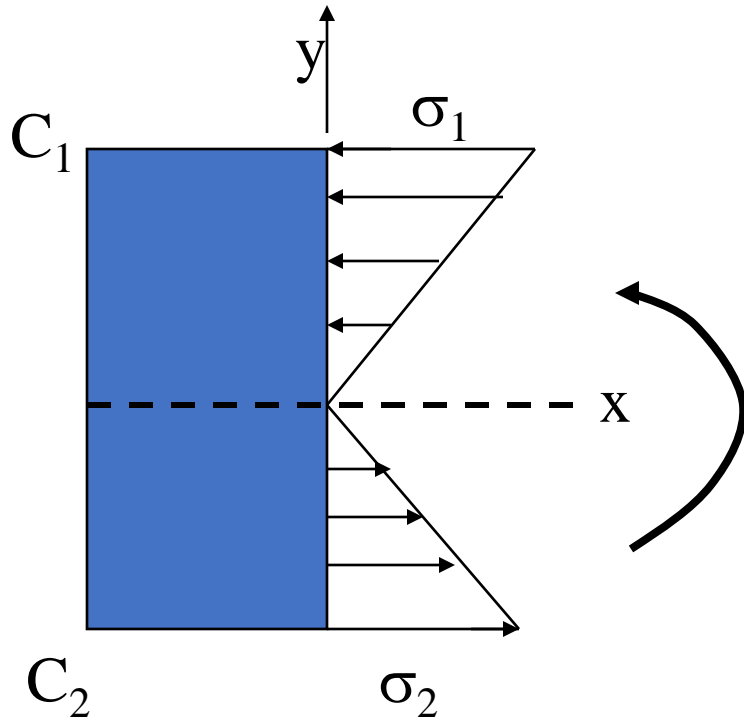
$$\sigma_x = -E \cdot \kappa \cdot y = -\frac{M \cdot y}{I}$$

MAXIMUM STRESSES

OCCUR AT  
THE TOP & BOTTOM FACES



# MAXIMUM STRESSES

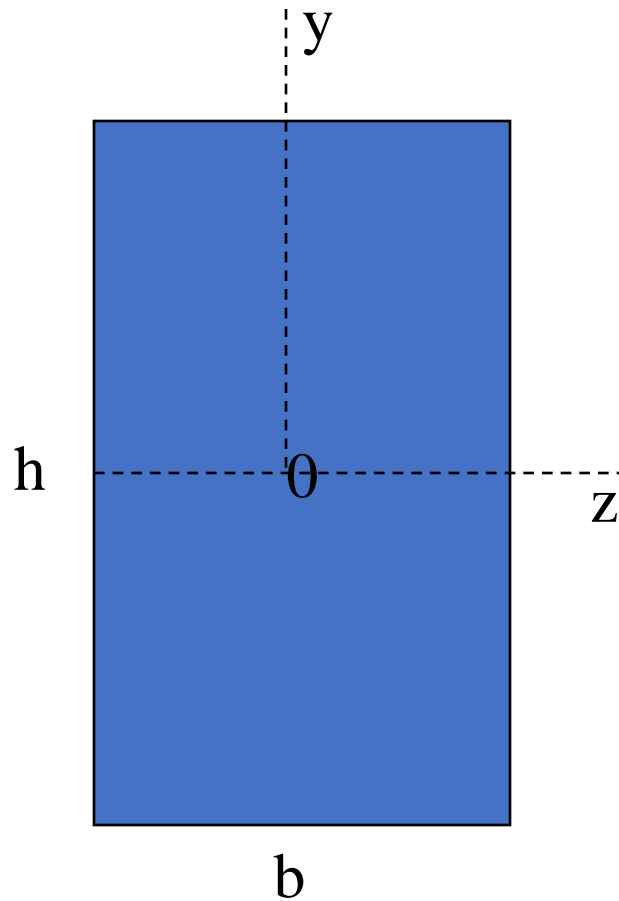


$$\sigma_1 = - \frac{M \cdot c_1}{I} = - \frac{M}{S_1}$$

$S_1 = I/c_1$   
Section modulus

$$\sigma_2 = - \frac{M \cdot c_2}{I} = - \frac{M}{S_2}$$

# I & S FOR BEAM



$$I = \frac{b \cdot h^3}{12}$$

$$S = \frac{b \cdot h^2}{6}$$

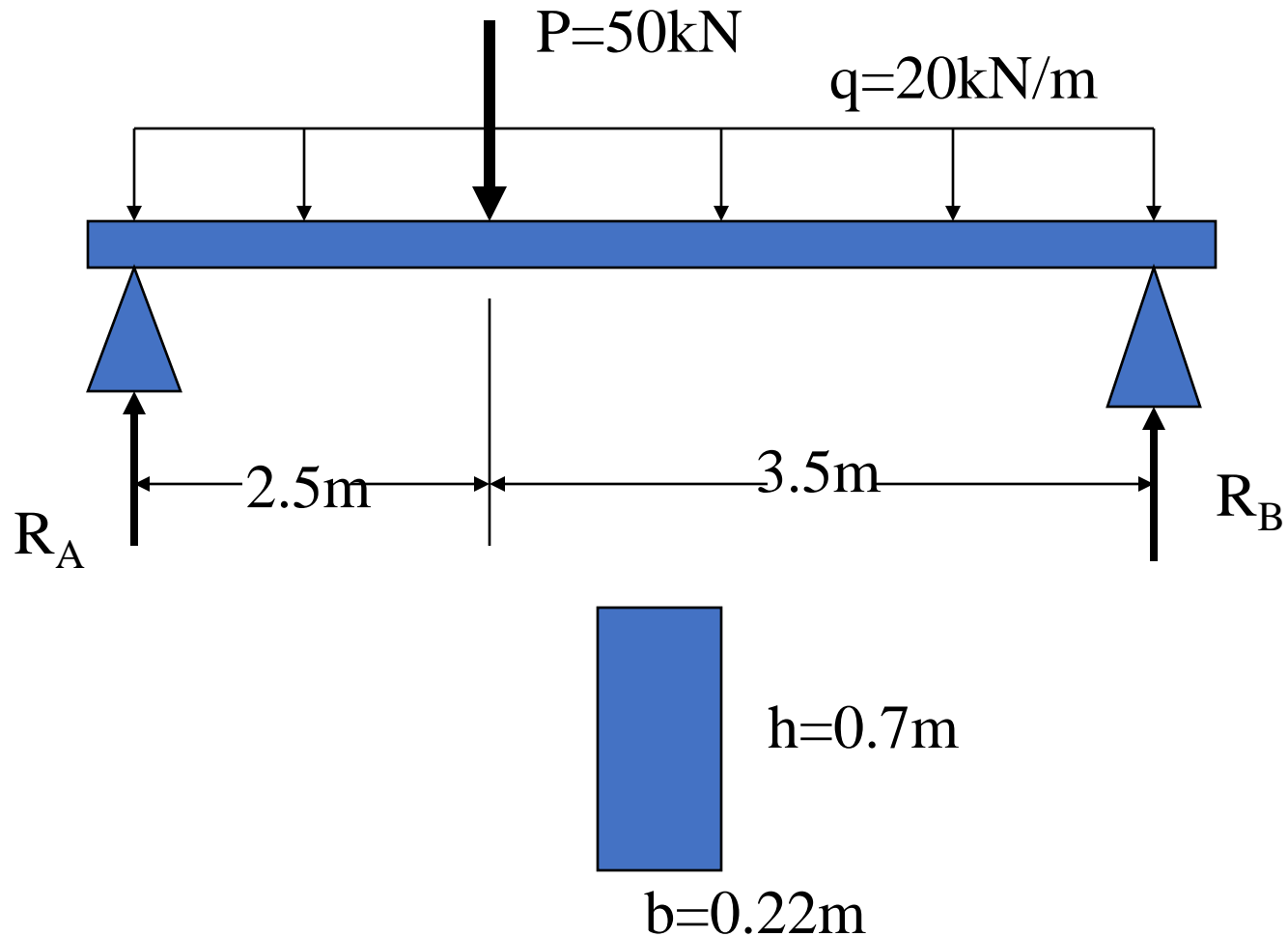
# BEAM DESIGN

- SELECT SHAPE AND SIZE SO THAT STRESS DOES NOT EXCEED  $\sigma_{\text{ALLOW}}$
- CALCULATE REQUIRED  $S = M_{\text{MAX}} / \sigma_{\text{ALLOW}}$
- CHOOSE LOWEST CROSS SECTION WHICH SATISFIES S

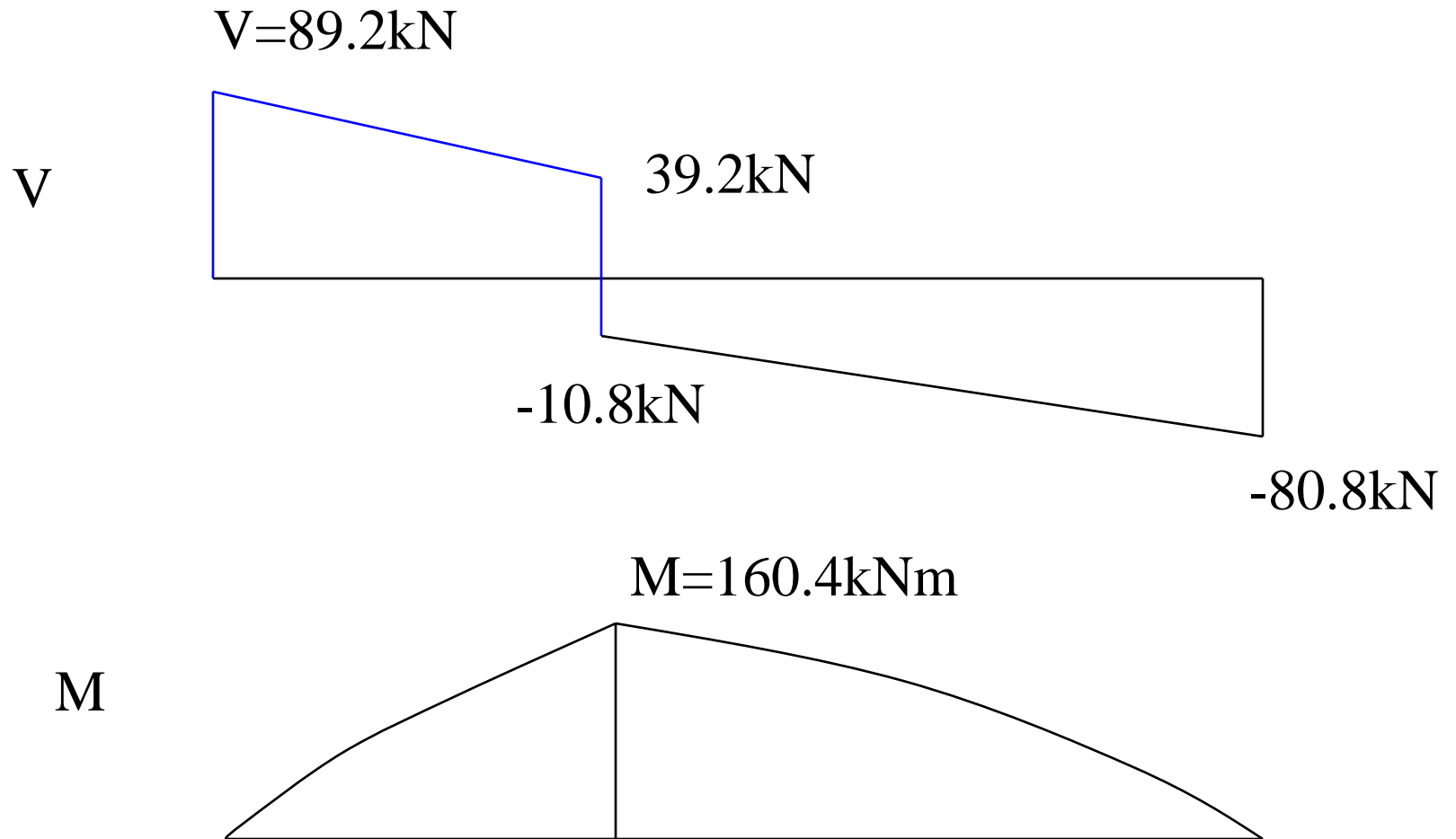
# IDEAL BEAM

- RECTANGULAR BEAM,  $S_R = bh^2/6 = Ah/6$
- CYLINDRICAL BEAM,  $S = 0.85.S_R$ 
  - IDEAL BEAM, HALF THE AREA AT  $h/2$
- IDEAL BEAM,  $S = 3.S_R$
- STANDARD I-BEAM,  $S = 2.S_R$

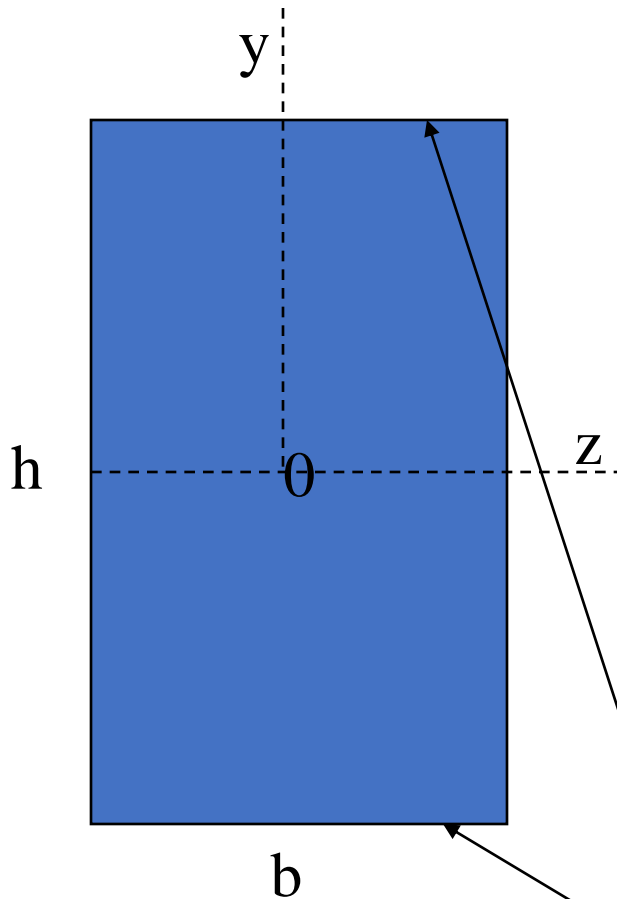
# STRESSES CAUSED BY BENDING



# V & M DIAGRAMS



$\sigma_{\text{MAX}}$  FOR BEAM



$$S = \frac{b \cdot h^3}{6} = \frac{0.22 \times 0.7^3}{6} = 0.018 \text{ m}^3$$

MAXIMUM STRESSES

$$\sigma_c = - \frac{M_{\text{max}}}{S} = - \frac{160.4 \text{ kNm}}{0.018 \text{ m}^3} = -8.9 \text{ MPa}$$

$$\sigma_t = +8.9 \text{ MPa}$$